

# Wellcome

- ▶ Achim Surber
- ▶ Study of mechanical engineering
- ▶ Study of business economics
- ▶ 30 years experience with pumps
- ▶ Head of sales department
- ▶ Hermetic Pumpen GmbH
- ▶ phone +49-761-5830-221
- ▶ [surber.achim@hermetic-pumpen.com](mailto:surber.achim@hermetic-pumpen.com)



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Chemnitz 1

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## ▶ Optimizing Ammonia Pump Performance

India is the world's largest producer of milk and the second-largest producer of fruit and vegetables.

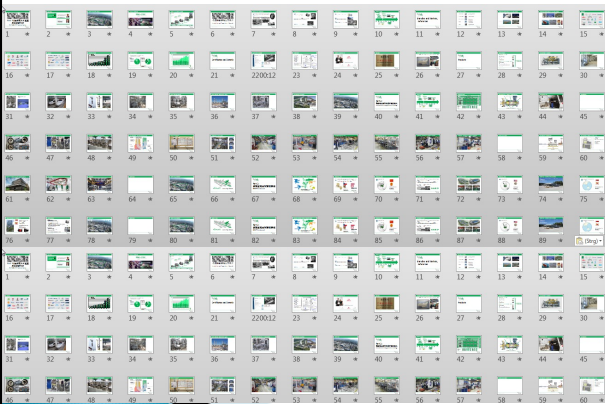


FoodNavigator  
AMMONIA REFRIGERATION

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### Overview of Slides 1- 37 / Achim Surber




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## AGENDA

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|    |  |     |
|----|--|-----|
| 01 | Types of pumps for Ammonia applicaton  | 10' |
| 02 | Optimizing performance - operation     | 15' |
| 03 | Common mistakes – pump installation    | 10' |
| 04 | Pump performance problems - Cavitation | 10' |



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### Pump is a very small but important part



Picture: Cooling system for modern 1800m3 tunar fishing vessel



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### Conventional pump with shaft seal

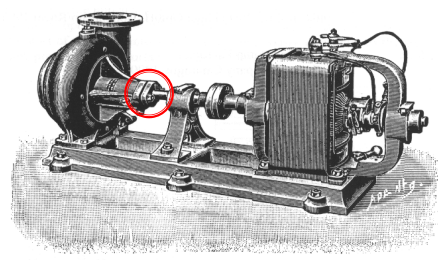
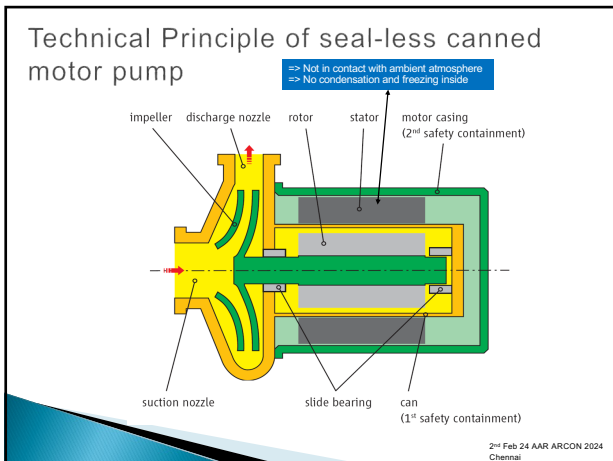


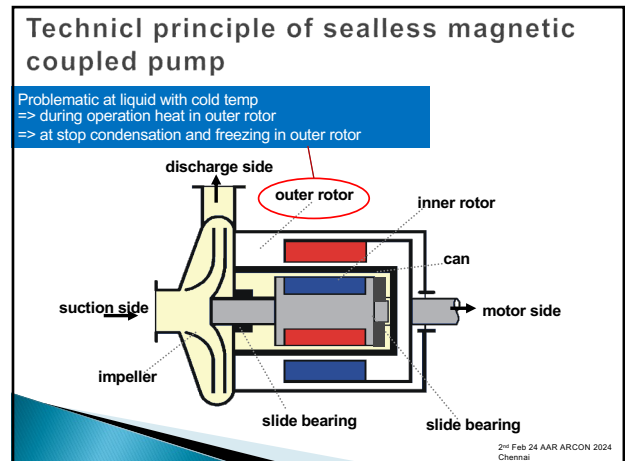
Fig 1.6 First standard production centrifugal pump from the W. Lederle Engine and Pump Factory from 1898.

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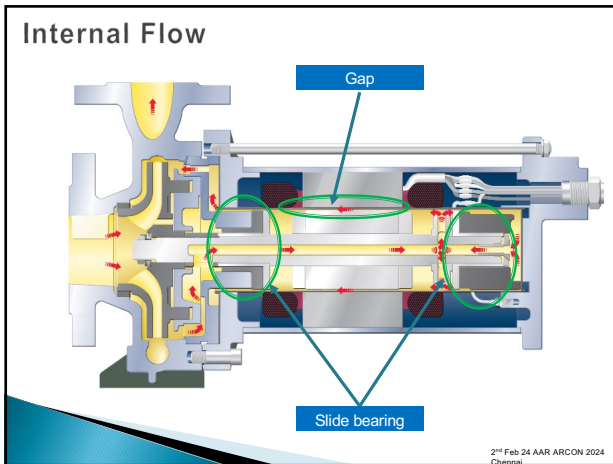
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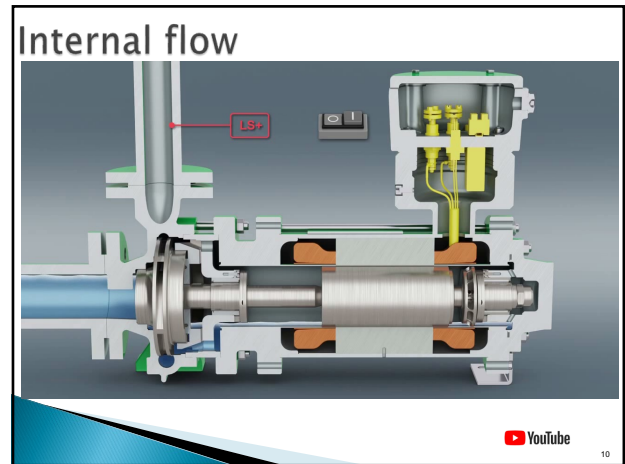
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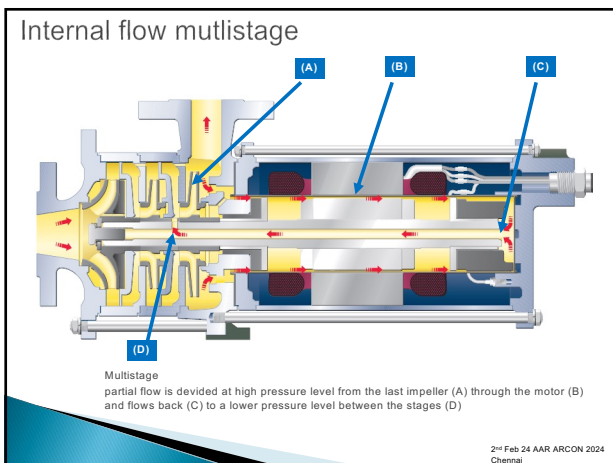
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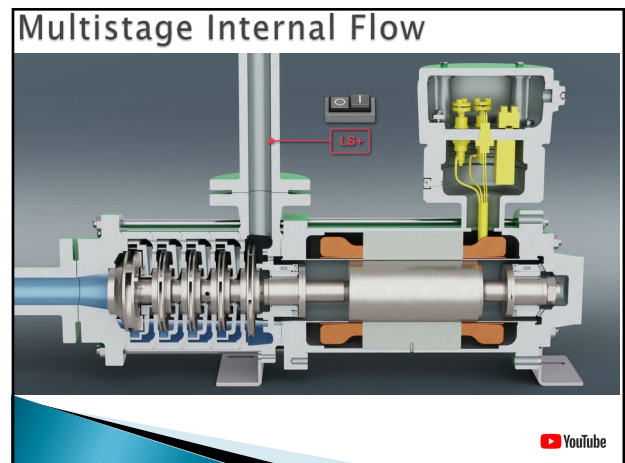
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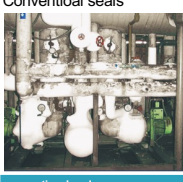
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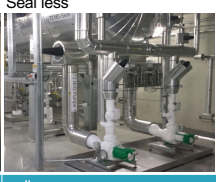
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### Technology compare

Conventional seals



Seal less



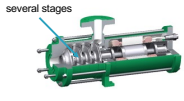
|                      | conventional seal                       | sealless  |
|----------------------|---|---|
| Pump efficiency      | 10-70%                                  | 10-70%  |
| Motor efficiency     | 90-97%                                  | appr. 85% because of can                        |
| Maintenance interval | 2-3 years (seals, roller ball bearings) | 8-10 years Up to 30 years at constant operation |
| Operational safety   | Leakage, mechanical seal                | Leakage free                                    |
| Seal failure         | Possible                                | Non seal  |

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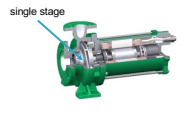
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### Technology compare

several stages



single stage

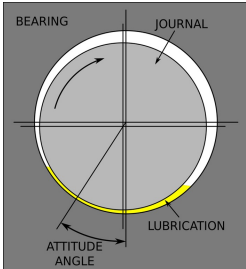


|                      | multi stage                          | single stage                      |
|----------------------|--------------------------------------|-----------------------------------|
| Pump efficiency      | Lower flow – high head               | Higher flow – lower head          |
| Typical Flow / Head  | 0.5-35m <sup>3</sup> /h / max. 130m  | 10-60m <sup>3</sup> /h / max. 55m |
| Head                 | By number of stages                  | By increase diameter              |
| Hydraulic efficiency | At higher head better                | Lower at higher head              |
| NPSH                 | Lower Often special suction impeller | higher                            |
| Vapor pressure       | High vapor pressure                  | Lower in vapor pressure handling  |

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### Technology slide bearing



**CENTRIFUGAL PUMPS MINIMUM OPERATING SPEED**

HYDRODYNAMIC SLIDE BEARINGS

**SPEED**

The shaft lift depends on:

- the rotation speed
- the lubricant viscosity
- the pressure on contact (weight)

A variable speed drive allows to optimize the speed (between 25 and 50 Hz). Subject to the centrifugal pumps minimum operating speed.

**LUBRICATION**

Canned motor pumps use the pumped medium as lubricant. The lubricants reduce friction, wear, seizing and help to cool the engine down.

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Sealless pumps should never run dry




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## AGENDA

24<sup>th</sup> January 2020 / Mumbai



AIRBORNE REFRIGERATION

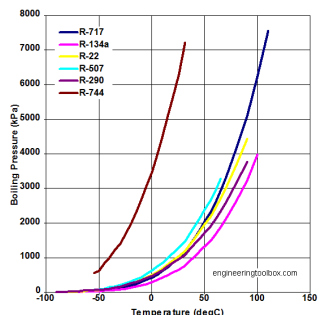
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- 04 Pump performance problems - Cavitation 10'

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### Refrigeration Liquid for pumping system

| Liquid       | Population |
|--------------|------------|
| R717 (NH3)   | 80%        |
| R22 (Frigen) |            |
| R744 (CO2)   | 12%        |
| R134a        |            |
| R11          |            |
| R12          |            |
| Propan       |            |
| Butan        |            |
| .....        |            |



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### Knowledge for correct Ammonia pump design

- Pressure loss pump entry NPSH
- Temperature increase hydraulic: Increase of vapor pressure
- Temperature increase drive: Risk of vaporization
- Backflow at wear ring 1<sup>st</sup> Impeller
- Pressure difference Impellers: Axial balance
- Hydrodynamic slide bearings: Lubrication and friction / viscosity

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### Software assisted selection of pump type

**Modern pump selection software Input:**

- Flow & head
- Liquid & temperature
- Refrigeration capacity

**Software calculation:**

- > 200 calculations in background
- Hydraulic and motor
- Thermal balance
- Vaporisation
- NPSH
- Min, max flow
- Data sheets
- GA drawing
- Sectional drawing

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### Installation and Operation principle

- Compressor
- Condenser
- Expansion valve
- Evaporator

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### Installation and Operation principle – min flow

**Why Q<sub>min</sub> orifice/bypass:**

- Continuous operating, even when expansion valve (1) at evaporator is closed
- Automatic venting when pump is stopped after operation

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### Installation and Operation principle – max flow

**Why Q<sub>max</sub> Protection:**

if all regulation valves are open, pressure ↓, flow ↑, NPSHR ↑  
Protecting against over flow

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### Installation and Operation principle – max flow

With orifice vs Without orifice

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### Operation

**Operation outside of allowable operation range**


Flow smaller than  $Q - \text{min}$ :

- => axial balance of pump disturbed
- => heating up of liquid,
- => vaporization in motor (mixed friction or dry running slide bearings)
- => increasing NPSH value



Exceeding  $Q - \text{max}$ :

- => high power consumption
- => increasing NPSH value
- => axial balance disturbed



**IMPORTANT**

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## AGENDA

24<sup>th</sup> January 2020 / Mumbai

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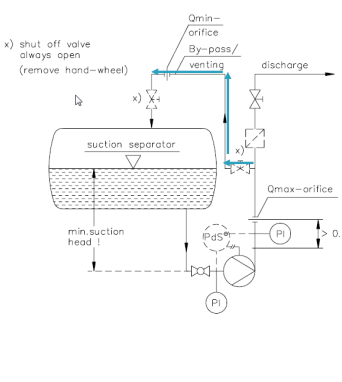
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### Installation and Operation - Venting

**Automatic venting of the pumps**

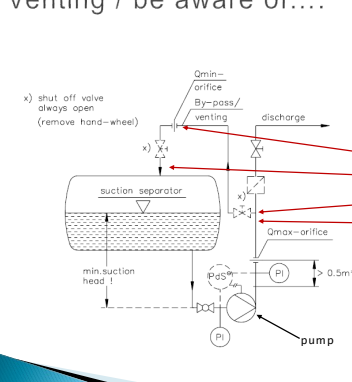
During standstill, the by-pass is used as a vent line. The pump should not be started unless it is primed with liquid refrigerant. During first priming or re-starting, wait until all pump parts have cooled down almost to the refrigerant temperature. When using non-return valves at the pump discharge, note that the by-pass line must be installed before the non-return valve, to permit automatic venting, **IMPORTANT!** No check valve should be used in the bypass line.



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### Venting / be aware of...



**Very Important for correct venting:**

- min orifice in gas phase
- end of min flow line in gas phase
- shut of valve of min flow line all time open
- min flow line before non return valve

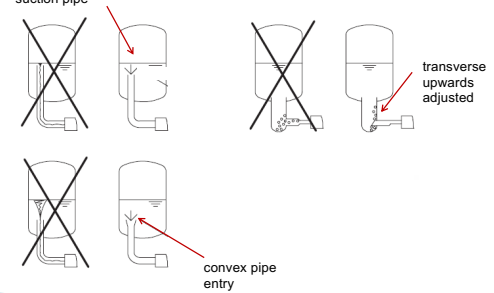
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### Failure caused suction conditions

Installation VESSEL

minimum level to suction pipe



transverse upwards adjusted

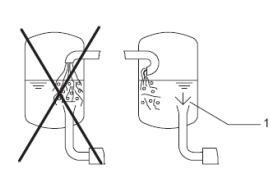
convex pipe entry

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### Failure caused suction conditions

Installation VESSEL



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### Failure caused suction conditions

Pressure lowering speed: pressure drop diagram

steep pressure drop curve

Boiling up of the liquid surface

Flat pressure drop curve

Liquid free of gas bubbles

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### Failure caused suction conditions

Installation suction pipe

Inlet sizing principle:  
L = 5'DNs  
Diameter pipe ≥ 2" Diameter suction nozzle  
Flow speed: 0,3-0,5m/s

wrong

correct

Suction pipe downwards

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### Suction Filter

- Suction filter is only for the commissioning period.
- mesh < 0,8mm
- Must be removed as soon as possible, or must be monitoring with Δpressure measurement

If necessary, install a sieve in the suction pipe only during the start-up phase

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### What is cavitation → Resons

- on the suction side - the static pressure of the flowing liquid falls below steam pressure
- bubbles occurs
- at areas of higher pressure steam bubbles will suddenly implode
- this imploding is the cause for very high short-term pressure peaks of up to some 1,000 bar

Steps in Cavitation

Formation of bubbles inside the liquid

Growth of bubbles

Collapse of bubbles

Cavitation

Fig. 2

www.chemrecor.com

Collapse of a vapor bubble

Fig. 11

Initial bubble

Detachment of bubble collapse

Forming of liquid microjet

Impact & metal extrusion

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### What is cavitation → Locations

at areas of higher pressure, steam bubbles will suddenly implode outer diameter / end of vanes

Pressure

4.73e+005

4.40e+005

4.26e+005

3.99e+005

3.74e+005

3.50e+005

3.25e+005

3.00e+005

2.75e+005

2.51e+005

2.26e+005

2.01e+005

1.76e+005

1.52e+005

1.27e+005

1.02e+005

7.72e+004


[Pa]

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### What is cavitation → Effects

pressure peaks of imploding bubbles lead to material erosions



damage of impeller



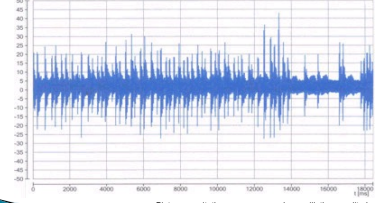
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### What is cavitation → Effects

**How we can deduct cavitation**

- We experience a loss in capacity
- We can no longer build the same head ( pressure drops, 3% value )
- The efficiency drops
- The cavities or bubbles will collapse when they pass into the higher regions of pressure causing noise, vibration



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### NPSH → Introduction

- **NPSH = Net Positive Suction Head**
- If the static pressure in the suction nozzle falls below the vapor pressure of the flowing liquid **CAVITATION** occurs.
- The **NPSH** value helps to describe and avoid such situation

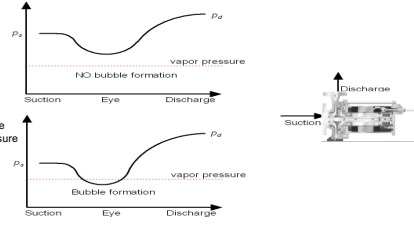
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### NPSH

**Two NPSH values:**  
 1.) NPSHr = NPSH required  
 2.) NPSHa = NPSH available

**NPSHr = NPSH value of the pump.**  
 When liquid flows into suction nozzle, the pressure falls shortly down. This pressure drop is described in the NPSHr value of pump.



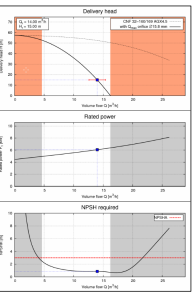
Ps = suction pressure  
 Pd = discharge pressure

**Pressure inside the pump must be all the time higher than the vapor pressure.**

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### NPSHa > NPSHr



| Design data:                          |             | Refrigerant:                            |                            |
|---------------------------------------|-------------|---|----------------------------|
| Refrigerating capacity Q <sub>0</sub> | 283.30 [kW] | Evaporation temperature T <sub>0</sub>  | -26.0 [°C]                 |
| Evaporation factor k                  | 4.0         | Density ρ <sub>0</sub>                  | 1.012 [kg/m <sup>3</sup> ] |
| Pressure drop Δp <sub>0</sub>         | 2.52 [bar]  | Specific heat capacity c <sub>p,0</sub> | 0.517 [kJ/kgK]             |
| NPSH available NPSHa                  | 3.00 [m]    | Vapour pressure p <sub>v,0</sub>        | 19.70 [bar]                |

| Operating point:             |                           | Equipment:                 |  |
|------------------------------|---------------------------|----------------------------|--|
| Volume flow Q <sub>0</sub>   | 14.00 [m <sup>3</sup> /h] | Q <sub>0, max</sub> effice |  |
| Delivery head H <sub>0</sub> | 15.00 [m]                 | Indicator                  |  |
| Net frequency f <sub>0</sub> | 60 [Hz]                   |                            |  |

| Nominal data:                         |                           | Drive:  |                           |
|---------------------------------------|---------------------------|---|---------------------------|
| Pump:                                 | CMF 32-110                | Rated input P <sub>1</sub>                      | 7.80 [kW]                 |
| Impeller diameter D <sub>2</sub>      | 126.0 [mm]                | Rated output P <sub>2</sub>                     | 5.40 [kW]                 |
| Construction:                         | BBH induction             | Frequency f <sub>1</sub>                        | 60 [Hz]                   |
| Volume flow Q <sub>0, nom</sub>       | 18.25 [m <sup>3</sup> /h] | Rated speed n <sub>1</sub>                      | 3340 [min <sup>-1</sup> ] |
| Delivery head H <sub>0, nom</sub>     | 48.02 [m]                 | Rated voltage U <sub>1</sub>                    | 276-276 [V]               |
| Shaft orifice diameter D <sub>2</sub> | 125.8 [mm]                | Rated current I <sub>N</sub>                    | 19.2 [A]                  |
|                                       |                           | Starting current I <sub>st</sub> I <sub>N</sub> | 2.7                       |
|                                       |                           | Power factor cos φ                              | 0.85                      |
|                                       |                           | Insulation class:                               | H                         |

| Operating range:             |                                  | Remarks:  |  |
|------------------------------|----------------------------------|---|--|
| Volume flow Q <sub>0</sub>   | 0.00 - 14.00 [m <sup>3</sup> /h] | ■ the attention to the advisable operating range as shown in the performance curve on page 2. |  |
| Delivery head H <sub>0</sub> | 5.27 - 15.00 [m]                 | ■ the attention to the advisable operating range as shown in the performance curve on page 2. |  |
| Net power P <sub>0</sub>     | 4.85 - 6.06 [kW]                 | ■ the pump selection will be checked  |  |
| NPSH required NPSHr          | 2.50 - 0.85 - 0.27 [m]           |   |  |


Picture: NPSHr value of pump in data sheet & in performance curve

**Theoretically NPSHa should be at least same level like NPSHr  
 For trouble free and safe operation:  
 NPSHa > NPSHr + 0.5m**

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### Conclusion



- 01 Sealless  
No leakage  
Less Maintenance
- 02 Installation  
Min & Max Flow  
Orifices
- 03 Guideline for Installation  
Venting  
Suction Conditions  
Filters
- 04 Cavitation  
Cause & Effect
- 05 NPSH  
NPSHa & NPSHr  
Safety Margin
- 06 End

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ARTICLE

**One-Third of Adults Have Snoozed During PowerPoint Presentations**



THANK YOU

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